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KINEMATICS, STATICS, AND DYNAMICS OF TWO-D MANIPULATORS

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ABSTRACT. In order to get some feeling for the kinematics, statics, and dynamics of manipulators, it is useful to separate the problem of visualizing linkages in three-space from the basic mechanics. The general-purpose two-dimensional manipulator is analyzed in this paper in order to gain a basic understanding of the issues without the complications of three-dimensional geometry.

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INTRODUCTION

Kinematics deals with the basic geometry of the linkages. If we consider an articulated manipulator as a device for generating position and orientation, we need to know the relationships between these quantities and the joint-variables, since it is the latter that we can easily measure and control. Position here refers to the position in space of the tip of the device, while orientation refers to the direction of approach of the last link. While position is fairly easy to understand in spaces of higher dimensionality, rotation or orientation becomes rapidly more complex. This is the main impetus for our study of two-dimensional devices. In two dimensions, two degrees of freedom are required to generate arbitrary positions in a given work-space and one more if we also want to control the orientation of the last link.

The first device studied in detail has only two joints and so can be used as a position generator. Later, a three-link device is discussed which is a general-purpose two-dimensional device that can generate orientation as well.

It will become apparent that the calculation of position and orientation of the last link given the joint-variables is straight-forward, while the inverse calculation is hard and may be intractable for devices with many links that have not been designed properly. This is important since the calculation of joint-angles given desired position and orientation is vital if the device is to be used to reach for objects, move them around or follow a given trajectory.

If a manipulator has just enough degrees of freedom to cover its work-space, there will in general be a finite number of ways of reaching a given position and orientation. This is because the inverse problem essentially corresponds to solving a number of equations in an equal number of unknowns. If the equations were linear we would expect exactly one solution.

Since they are trigonometric polynomials in the joint-variables - and hence non-linear - we expect a finite number of solutions. Similarly, if we have too few joints, there will in general be no solution, while with too many joints we expect an infinite number of ways of reaching a given position and orientation.

Usually there are some arm configurations that present special problems because the equations become singular. These often occur on the boundary of the work-space, where some of the links become parallel.

Statics deals with the balance of forces and torques required when the device does not move. If we consider an articulated manipulator as a device for applying forces and torques to objects being manipulated, we need to know the relationship between these quantities and the joint-torques, since it is the latter that we either directly control or at least can measure. In two dimensions, two degrees of freedom will be required to apply an arbitrary force at the tip of the device and one more if we want to control torque applied to the object as well.

Clearly then the two-link device to be discussed can be thought of as a force generator, while the three-link device can apply controlled torques as well. The gravity loading of the links has to be compensated for as well and fortunately it can be considered separately from the the torques required to produce tip forces and torques.

Dynamics, finally, deals with the manipulator in motion. It will be seen that the joint-torques control the angular accelerations. The relationships are not direct however. First of all, the sensitivity of a given joint to torque varies with the arm-configuration, secondly, forces appear that are functions of the products of the angular veloctles and thirdly there is considerable coupling between the motions of the links. The velocity product terms can be thought of as generalized centrifugal forces.

The equations relating joint-accelerations to joint-torques are non-linear of course, but given the arm-state - that is both joint-variables and their rate of change with time - it is straight-forward to calculate what joint-torques are required to achieve given angular accelerations. We can in other words, calculate the time-history of motor-torques for each joint required to cause the arm to follow a given trajectory.

Notice that this is an open-loop dead-reckoning approach which in practice has to be modified to take into account friction and small errors in estimating the numerical constant in the sensitivity matrix. The modification can take the form of a small amount of compensating feed-back.

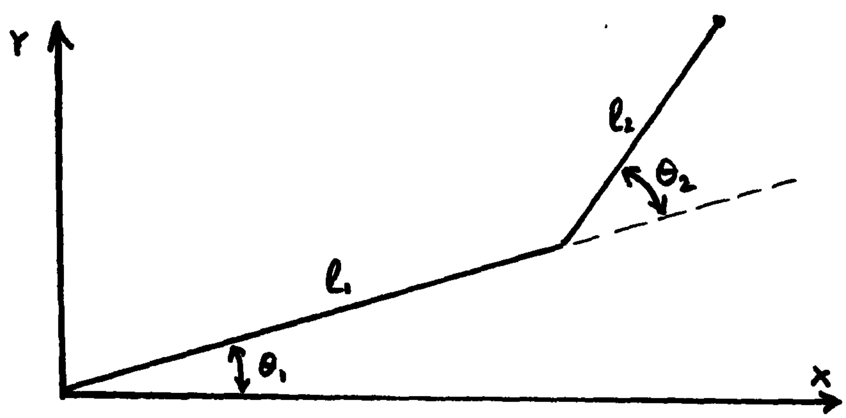
This however should not be confused with the more traditional, analog servo methods which position-controls each joint independently and cannot deal properly with the dynamics at all.

To summarize: we will deal with unconstrained motion of the manipulator as it follows some trajectory as well as its interaction with parts that mechanically constrain its motion. Both aspects of manipulator operation are of importance if it is to be used to assemble or disassemble artifacts.

TWO-LINK MANIPULATOR

In order to get some feeling for the kinematics, statics and dynamics of articulated manipulators it is helpful to start by studying some stripped-down versions. In particular if we confine operations to two dimensions, sketches and geomtric insight are more readily produced. In two dimensions one clearly needs two degrees of freedom to reach an arbitrary point within a given work-space. Let us first study a simple two-link manipulator with rotational joints. Devices with extensional joints are even simpler, but do not illuminate many of the important issues. Note also that the geometry of the two-link device occurs as a sub-problem in many of the more complicated manipulators.

KINEMATICS

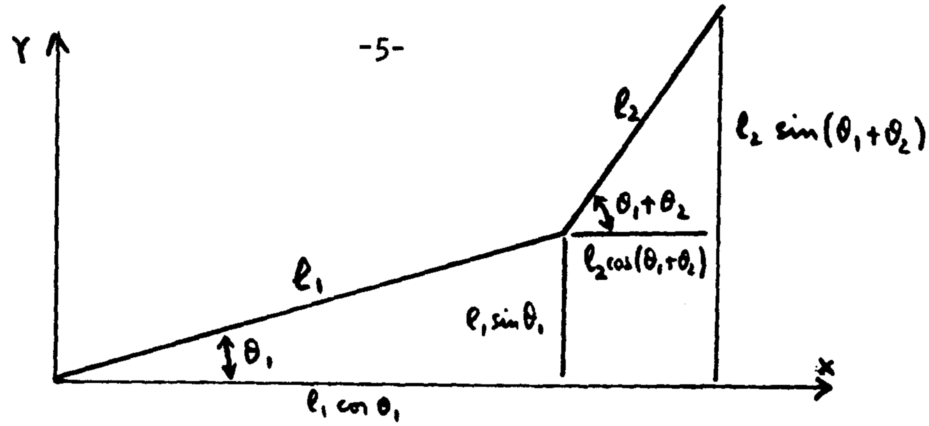


Given the two joint-angles, let us calculate the position of the tip of the device. Define vectors corresponding to the two links:

$$\underline{r}_1 = l_1 \; (\cos(\theta_1) \; , \; \sin(\theta_1) \;)$$

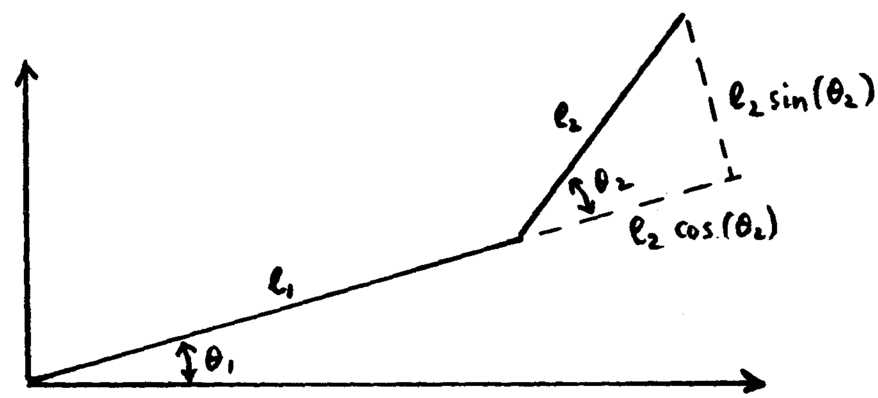
$$\underline{r}_2 = l_2 \; (\cos(\theta_1+\theta_2) \; , \; \sin(\theta_1+\theta_2) \;)$$

Then the position of the tip $\underline{r}$ can be found simply by vector addition.



$$x = l_1 \cos(\theta_1) + l_2 \cos(\theta_1+\theta_2)$$
$$y = l_1 \sin(\theta_1) + l_2 \cos(\theta_1+\theta_2)$$

This can be expanded into a slightly more useful form:



$$x = [l_1 + l_2 \cos(\theta_2)] \cos(\theta_1) - l_2 \sin(\theta_2) \quad \sin(\theta_1)$$
$$y = [l_1 + l_2 \cos(\theta_2)] \sin(\theta_1) + l_2 \sin(\theta_2) \quad \cos(\theta_1)$$

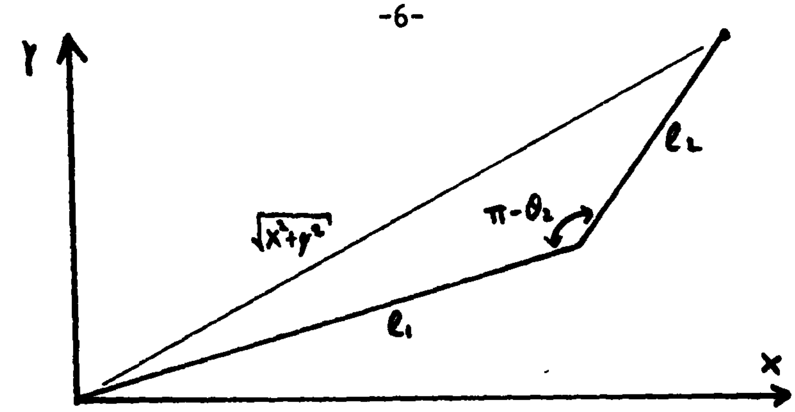
POLAR COORDINATES:

Because of the rotational symmetry of the device about the origin it may be more natural to think of it as a " r, θ Generator" than a " x, y Generator". We expect for example that r will not depend on θ_1. To calculate r we can proceed along various avenues:

$$|\underline{r}|^2 = |\underline{r}_1+\underline{r}_2|^2 = |\underline{r}_1|^2 + 2\, \underline{r}_1 \cdot \underline{r}_2 + |\underline{r}_2|^2$$

Or we can use the expressions for x and y:

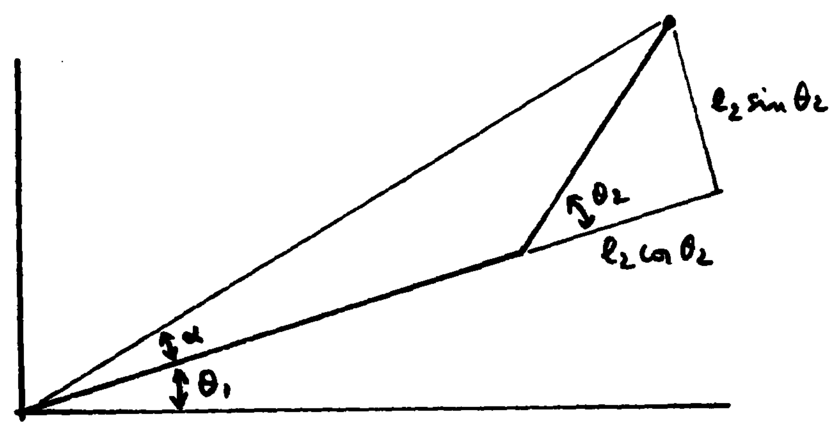
$$x^2 + y^2 = [l_1 + l_2 \cos(\theta_2)]^2 + [l_2 \sin(\theta_2)]^2$$



We can also use a formulae for the solution of triangles. In each case we get:

$$r^2 = x^2 + y^2 = l_1^2 + 2\, l_1\, l_2 \cos(\theta_2) + l_2^2$$

Next we have to find θ. We can proceed from $\tan(\theta) = y/x$ and use the expressions for x and y or consider the following sketch:



Where $\theta = \theta_1 + \alpha$ and $\tan(\alpha) = l_2 \sin(\theta_2) \,/\, [l_1 + l_2 \cos(\theta_2)]$. Using the formula for expanding tangent of the sum of two angles one gets:

$$\tan(\theta) = \frac{[l_1 + l_2 \cos(\theta_2)] \sin(\theta_1) + l_2 \sin(\theta_2) \cos(\theta_1)}{[l_1 + l_2 \cos(\theta_2)] \cos(\theta_1) - l_2 \sin(\theta_2) \sin(\theta_1)}$$

We now have expressions that allow us to calculate coordinates generated by given joint-angles for both cartesian and polar coordinate systems.

THE INVERSE PROBLEM:

When one uses a manipulator one is more interested in calculating the joint-angles that will place the tip of the device in some desired position. While the forward calculation of tip-position from joint-angles is always relatively straight-forward, the inversion is intractable for manipulators with more than a few links unless the device has been specially designed with this problem in mind.

For our simple device we easily get:

$$\cos(\theta_2) = \frac{(x^2 + y^2) - (l_1^2 + l_2^2)}{2\, l_1\, l_2}$$

There will be two solutions for θ_2 of equal magnitude and opposite sign. Expanding $\tan(\theta_1) = \tan(\theta - \alpha)$ and using $\tan(\theta) = y/x$ we also arrive at:

$$\tan(\theta_1) = \frac{-\, l_2 \sin(\theta_2)\ x + [l_1 + l_2 \cos(\theta_2)]\, y}{l_2 \sin(\theta_2)\ y + [l_1 + l_2 \cos(\theta_2)]\, x}$$

Where it may be useful to know that $l_1 + l_2 \cos(\theta_2) = [(x^2+y^2) + (l_1^2-l_2^2)] / (2 l_1)$

The reason this was so easy is that we happened to have already derived all the most useful formula using geometric and trigonometric reasoning. A method of more general utility depends on algebraic manipulation of the expressions for the coordinates of the tip. Notice that these expressions are polynomials in the sines and cosines of the joint-angles. Such systems of polynomials can be solved systematically - unfortunately the degree of the intermediate terms grows explosively as more and more variables are eliminated. So this method, while quite general, is in practice limited to solving only simple linkages.

Let us apply it to our two-link device.

$$x = [l_1 + l_2 \cos(\theta_2)] \cos(\theta_1) - l_2 \sin(\theta_2) \sin(\theta_1)$$
$$y = [l_1 + l_2 \cos(\theta_2)] \sin(\theta_1) + l_2 \sin(\theta_2) \cos(\theta_1)$$

We have already seen that adding the square of the equation for x and the square of the equation for y eliminates terms in θ_1.

$$x^2 + y^2 = l_1^2 + 2\, l_1\, l_2 \cos(\theta_2) + l_2^2$$

Next, note that the form of the equations suggests a rotation by θ_1 - applying the inverse rotation one gets:

$$x \cos(\theta_1) + y \sin(\theta_1) = l_1 + l_2 \cos(\theta_2)$$
$$-x \sin(\theta_1) + y \cos(\theta_1) = l_2 \sin(\theta_2)$$

It is easy to solve this pair of linear equations for $\sin(\theta_1)$ and $\cos(\theta_1)$.

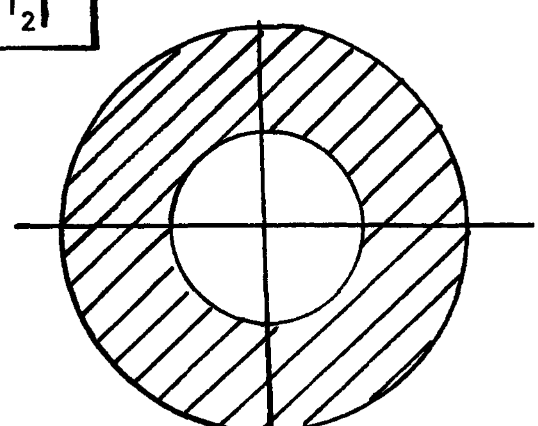
$$\sin(\theta_1) = \{- l_2 \sin(\theta_2) x + [l_1 + l_2 \cos(\theta_2)] y\} / (x^2+y^2)$$
$$\cos(\theta_1) = \{ l_2 \sin(\theta_2) y + [l_1 + l_2 \cos(\theta_2)] x\} / (x^2+y^2)$$

You may now notice that we could have solved the original equations for $\sin(\theta_1)$ and $\cos(\theta_1)$ in a similar fashion. The result would have been the same with the term $(l_1^2 + 2\, l_1\, l_2 \cos(\theta_2) + l_2^2)$ appearing in place of x^2+y^2.

THE WORK-SPACE:

$$(l_1 - l_2)^2 \leq l_1^2 + 2\, l_1\, l_2 \cos(\theta_2) + l_2^2 \leq (l_1 + l_2)^2$$

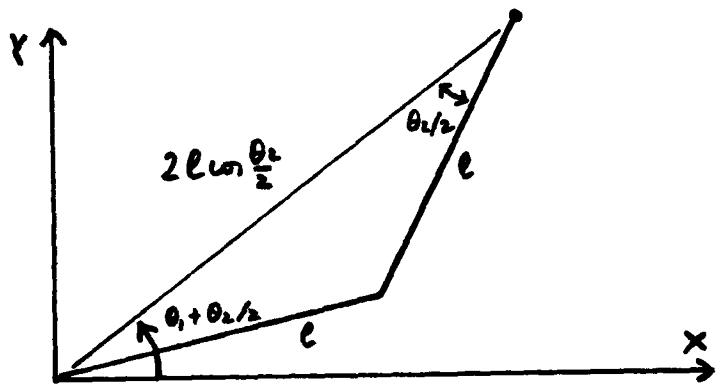
So: $$\boxed{|l_1 - l_2| \leq \sqrt{x^2 + y^2} \leq |l_1 + l_2|}$$



The set of points reachable by the tip of the device is an annulus centered on the origin. Notice that points on the boundary of this region can be reached in one, way, while points inside can be reached in two. The width of the annulus is twice the length of the shorter link and its average radius equals the length of the longer one.

EQUAL LINKS:

When $l_1 = l_2 = l$ say, the work-space becomes simpler, just a circle. The origin is a singular point in that it can be reached in an infinite number of ways - since θ_1 can be chosen freely. Equal link length provides some further simplification as well as an improved work-space geometry.



$$x = 2\ l\ \cos(\theta_2/2)\ \cos(\theta_1 + \theta_2/2)$$
$$y = 2\ l\ \cos(\theta_2/2)\ \sin(\theta_1 + \theta_2/2)$$

So

$$x^2 + y^2 = 2\ l^2\ (1 + \cos(\theta_2)) = 4\ l^2\ \cos^2(\theta_2/2)$$
$$\theta = \theta_1 + \theta_2/2$$

The inversion is solved as follows:

$$\cos(\theta_2) = (x^2 + y^2)/(2\ l^2) - 1$$

or

$$\cos(\theta_2/2) = \sqrt{x^2 + y^2}\ /(2\ l)$$

$$\tan(\theta_1 + \theta_2/2) = y/x$$

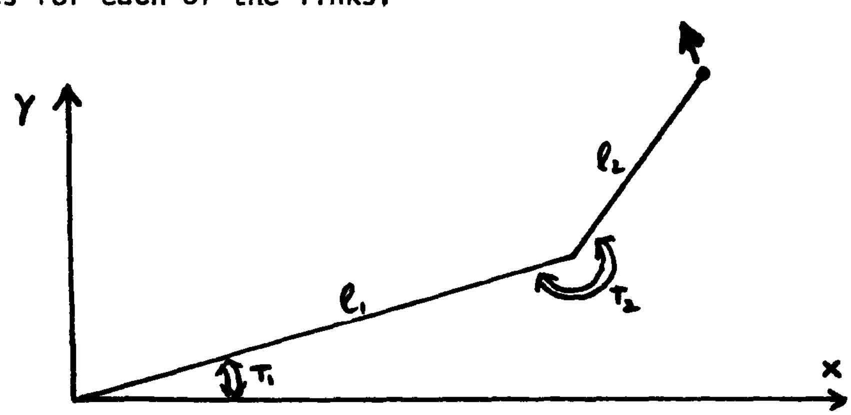
or

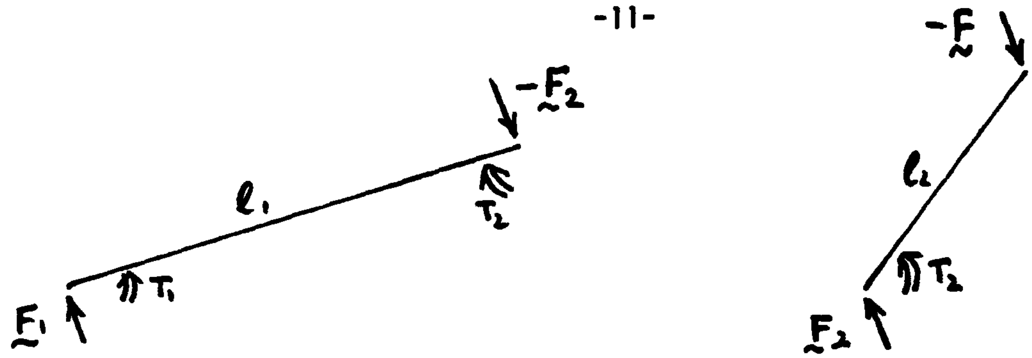
$$\tan(\theta_1) = \frac{-\sin(\theta_2/2)x + \cos(\theta_2/2)y}{\cos(\theta_2/2)x + \sin(\theta_2/2)y}$$

STATICS

So far we have thought of the manipulator as a device for placing the tip in any desired position within the work-space - that is, a position generator. Equally important is the device's ability to exert forces on objects. Let us assume that the manipulator does not move appreciably when used in this way so that we can ignore torques and forces used to accelerate the links. Initially we will also ignore gravity - we will later calculate the additional torques required to balance gravity components.

We have direct control over the torques T_1 and T_2 generated by the motors driving the joints. What forces are produced by these torques at the tip? Since we do not want the device to move, imagine its tip pinned in place. Let the force exerted by the tip on the pin be $\underline{F} = (u,v)$. To find the relationships between the forces at the tip and the motor torques, we will write down one equation for balance of forces and one equation for balance of torques for each of the links.





Note that the forces applied to each of the pin-joints in the device also have to balance as indicated in the diagram.

Writing down the equations for balance of forces in each of the two links we get:

$$\underline{F}_1 = \underline{F}_2 \quad \text{and} \quad \underline{F}_2 = \underline{F} \qquad \text{that is} \quad \underline{F} = \underline{F}_1 = \underline{F}_2$$

Next, picking an arbitrary axis for each of the links we get the equations for balance of torques:

$$T_1 - T_2 = \underline{r}_1 \times \underline{F}$$
$$T_2 = \underline{r}_2 \times \underline{F}$$

Where (a,b) x (c,d) = ad-bc. Adding the two equations one gets:

$$T_1 = (\underline{r}_1 + \underline{r}_2) \times \underline{F}$$

Right away we can tell what force components will be generated by each torque acting on its own. If $T_2 = 0$, then $\underline{r}_2 \times \underline{F} = 0$ and so $\underline{r}_2$ and $\underline{F}$ must be parallel, while $T_1 = 0$, gives $(\underline{r}_1 + \underline{r}_2) \times \underline{F} = 0$ and $(\underline{r}_1 + \underline{r}_2)$ is parallel to $\underline{F}$. These directions for $\underline{F}$ are counter-intuitive if anything!

Expanding the cross-products we get:

$$T_1 = [l_1 \cos(\theta_1) + l_2 \cos(\theta_1+\theta_2)]\, v - [l_1 \sin(\theta_1) + l_2 \sin(\theta_1+\theta_2)]\, u$$
$$T_2 = [l_2 \cos(\theta_1+\theta_2)]\, v - [l_2 \sin(\theta_1+\theta_2)]\, u$$

Using these results we can easily calculate what torques the motors should apply at the joints to produce a desired force at the tip.

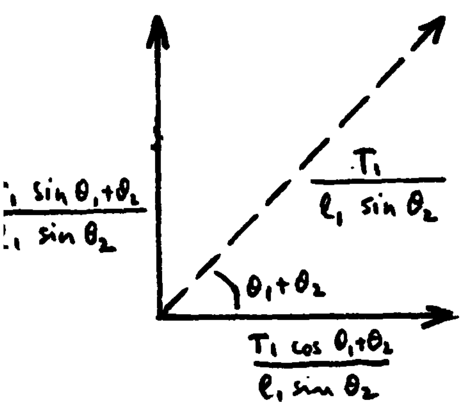
THE INVERSE PROBLEM

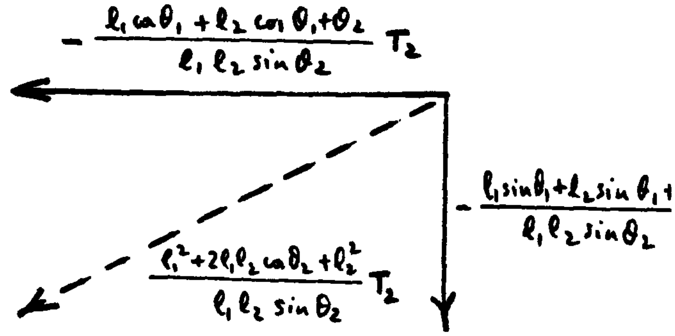
Now suppose we want to invert this process - calculate the force at the tip given measured joint-torques. Fortunately this inversion is straight-forward, we simply solve the pair of equations for u and v:

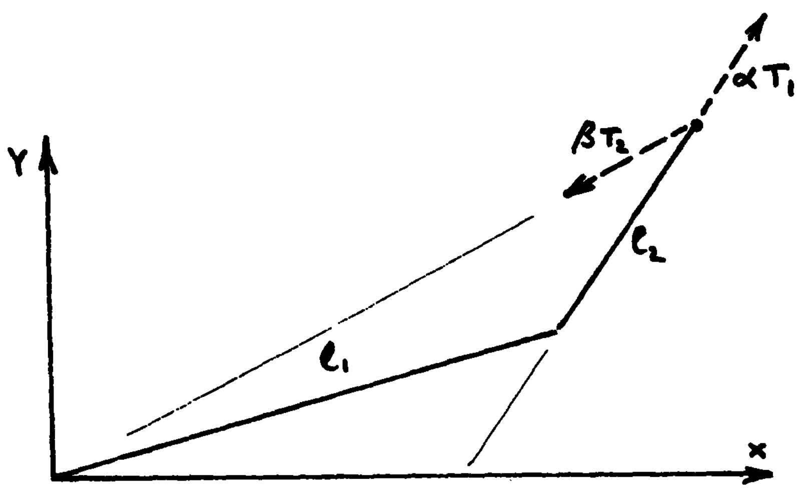
$$u = \{ l_2 \cos(\theta_1+\theta_2) \quad T_1 - [l_1 \cos(\theta_1) + l_2 \cos(\theta_1+\theta_2)] \, T_2 \} / (l_1 \; l_2 \sin(\theta_2))$$

$$v = \{ l_2 \sin(\theta_1+\theta_2) \quad T_1 - [l_1 \sin(\theta_1) + l_2 \sin(\theta_1+\theta_2)] \, T_2 \} / (l_1 \; l_2 \sin(\theta_2))$$

Now we can see in quantitative terms the force components produced by each joint-torque acting on its own:



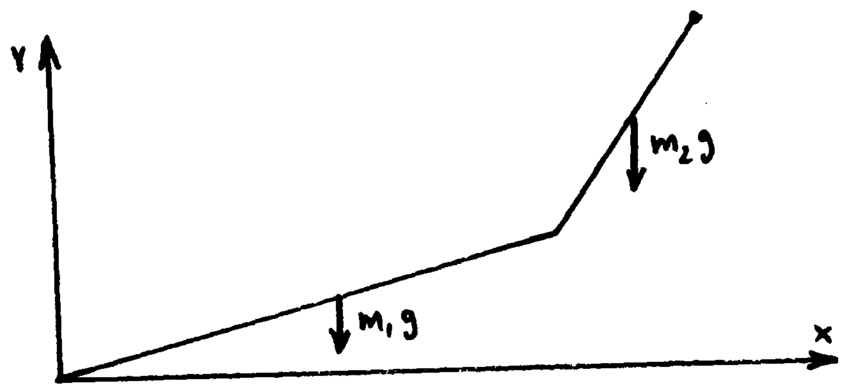




There are singularities in the transformation when $\sin(\theta_2) = 0$, that is when $\theta_2 = 0$ or π. Obviously when the links are parallel, the joint-torques have no control over the force component along the length of the links. Again we see the special nature of the boundary of the work-space.

GRAVITY

Let us assume for concreteness that the center of mass of each link is at its geometric center and let us define a gravity vector $\underline{g} = (0,-g)$ acting in the negative y direction. We could now repeat the above calculation with two additional components in the force-balance equations due to the gravity loading. Inspection of the equations shows that the resultant torques are linear in the applied forces, so we can use the principle of superposition, and calculate the gravity induced torques separately.



When there is no applied force at the tip we find that $\underline{F}_2 = m_2\underline{g}$ and $\underline{F}_1 = \underline{F}_2 + m_1\underline{g} = (m_1+m_2)\underline{g}$. Considering the torques we find:

$$T_{2g} = - m_2 \tfrac{1}{2}\underline{r}_2 \times \underline{g} = g\left[\tfrac{1}{2}m_2 l_2 \cos(\theta_1+\theta_2)\right]$$

$$T_{1g} = T_{2g} - m_1 \tfrac{1}{2}\underline{r}_1 \times \underline{g} - m_2 \underline{r}_1 \times \underline{g}$$

$$= g\left[(\tfrac{1}{2}m_1+m_2) l_1 \cos(\theta_1) + \tfrac{1}{2}m_2 l_2 \cos(\theta_1+\theta_2)\right]$$

These terms can now be added to the torque terms derived earlier for balancing the force applied at the tip.

In the next section we will remove the pin holding the tip in place and see how the device moves when torques are applied to the joints.

DYNAMICS:

Now let us determine what happens if we apply torques to the joints. What angular accelerations of the links will be produced? Knowing the relation between these two quantities will allow us to control the motions of the device as it follows some desired trajectory. We could proceed along lines similar to the ones followed when we studied statics, simply adding Newton's law.

$$\underline{F} = m\underline{a} \qquad \text{or} \qquad T = I\alpha$$

Where $\underline{F}$ is a force, m mass and $\underline{a}$ linear acceleration, Similarly T is a torque, I moment of inertia and α angular acceleration. The quantities involved would have to be expresses relative to some cartesian coordinate system. We would be faced with large sets of non-linear equations, since the mechanical constraints introduced by the linkage would have to be explicitly included and the coordinates of each joint expressed. In general this method becomes quite unwieldy for manipulators with more than a few links. The more general form of Newton's law gives a hint as to how one might proceed instead.

$$F_i = \frac{d}{dt}(mv_i)$$

F_i is a component of the force and mv_i is a component of the linear momentum. It is possible to develop a similar equation in a generalized coordinate system, that does not have to be cartesian. It is natural to chose the joint-angles as the generalized coordinates. These provide a compact description of the arm-configuration and the mechanical constraints are implicitely taken care of. It can be shown that:

$$Q_i = \frac{d}{dt}\,p_i - \frac{\partial L}{\partial q_i}$$

Where Q_i is a generalized force, p_i generalized momentum and q_i one of the generalized coordinates. There is one such equation for each degree of freedom. Q_i will be a force for an extensional joint, and a torque for a rotational joint - $Q_i q_i$ always has the dimensions of work.

LAGRANGES EQUATION

In this relation, L is the Lagrangian or "kinetic potential", equal to the difference between kinetic and potential energy, K - P. The generalized momentum p_i can be expressed in terms of L:

$$p_i = \frac{\partial L}{\partial \dot{q}_i}$$

This is analogous to $mv = \frac{d}{dv}(\frac{1}{2}mv^2)$. The dot represents differentiation with respect to time. Finally:

$$\boxed{\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{q}_i}\right) - \frac{\partial L}{\partial q_i} = Q_i}$$

And once again there is one such equation for each degree of freedom of the device. The next thing to remember is that the kinetic energy of a rigid body can be decomposed into a component due to the instantaneous linear translation of its center of mass ($\frac{1}{2}mv^2$) and a component due to the instantaneous angular velocity ($\frac{1}{2}I\omega^2$).

It will be convenient to ignore gravity on the first round - so there will be no potential energy term. Next we will take the simple case of equal links and let the links be sticks of equal mass m and uniform mass distribution. The moment of inertia for rotation about the center of mass of such a stick is $(1/12)ml^2$. These assumptions allow a great deal of simplification of intermediate terms without loosing much of importance. In fact the final result would be the same, except for some numerical constants if we had considered the more general case.

In order to calculate kinetic energy we will need the linear and angular velocities of the links. The angular velocities obviously are just $\dot{\theta}_1$ and $(\dot{\theta}_1+\dot{\theta}_2)$. The magnitudes of the instantaneous linear velocities of the center of mass are:

$$\left|\tfrac{1}{2}\, \underline{r}_1\, \dot{\theta}_1\right| \quad \text{and} \quad \left|\underline{r}_1\dot{\theta}_1 + \tfrac{1}{2}\, \underline{r}_2(\dot{\theta}_1+\dot{\theta}_2)\right|$$

The squares of these quantities are:

$$\tfrac{1}{4}\, l^2\, \dot\theta_1^2 \quad \text{and} \quad l^2 \left[\dot\theta_1^2 + \cos(\theta_2)\, \dot\theta_1(\dot\theta_1+\dot\theta_2) + \tfrac{1}{4}\, (\dot\theta_1+\dot\theta_2)^2 \right]$$

The total kinetic energy of link 1 is then:

$$\tfrac{1}{2}(\tfrac{1}{12})ml^2\, \dot\theta_1^2 + \tfrac{1}{2}\, m\, \tfrac{1}{4}\, l^2\, \dot\theta_1^2 = \tfrac{1}{2}(\tfrac{1}{3}ml^2)\dot\theta_1^2$$

The same result could have been obtained more directly by noting that the moment of inertia of a stick about one of its ends is $(1/3)ml^2$.

The total kinetic energy of link 2 is:

$$\tfrac{1}{2}(\tfrac{1}{12}ml^2)(\dot\theta_1+\dot\theta_2)^2 + \tfrac{1}{2}ml^2 \left[(\tfrac{5}{4} + \cos(\theta_2))\dot\theta_1^2 + (\tfrac{1}{2} + \cos(\theta_2))\dot\theta_1\dot\theta_2 + \tfrac{1}{4}\dot\theta_2^2 \right]$$

$$= \tfrac{1}{2}ml^2 \left[(\tfrac{4}{3} + \cos(\theta_2))\, \dot\theta_1^2 + (\tfrac{2}{3} + \cos(\theta_2))\, \dot\theta_1\dot\theta_2 + \tfrac{1}{3}\, \dot\theta_2^2 \right]$$

Finally, adding all components of the kinetic energy and noting that $P = 0$,

$$\boxed{L = \tfrac{1}{2}ml^2 \left[(\tfrac{5}{3} + \cos(\theta_2))\, \dot\theta_1^2 + (\tfrac{2}{3} + \cos(\theta_2))\, \dot\theta_1\dot\theta_2 + \tfrac{1}{3}\, \dot\theta_2^2 \right]}$$

Next we will need the partial derivatives of L with respect to θ_1, θ_2, $\dot\theta_1$ and $\dot\theta_2$. For convenience let $L = \tfrac{1}{2}ml^2\, L'$.

$$\frac{\partial L'}{\partial \theta_1} = 0$$

$$\frac{\partial L'}{\partial \theta_2} = -\sin(\theta_2)\, \dot\theta_1(\dot\theta_1+\dot\theta_2)$$

$$\frac{\partial L'}{\partial \dot\theta_1} = 2\, (\tfrac{5}{3} + \cos(\theta_2))\, \dot\theta_1 + (\tfrac{2}{3} + \cos(\theta_2))\dot\theta_2$$

$$\frac{\partial L'}{\partial \dot\theta_2} = (\tfrac{2}{3} + \cos(\theta_2))\, \dot\theta_1 + 2\, (\tfrac{1}{3})\, \dot\theta_2$$

We will also require the time derivatives of these last two expressions:

$$\frac{d}{dt}\left(\frac{\partial L'}{\partial \dot{\theta}_1}\right) = \ddot{\theta}_1\ 2\ \left(\frac{5}{3} + \cos(\theta_2)\right) + \ddot{\theta}_2\ \left(\frac{2}{3} + \cos(\theta_2)\right) - \sin(\theta_2)\ \dot{\theta}_2(2\dot{\theta}_1+\dot{\theta}_2)$$

$$\frac{d}{dt}\left(\frac{\partial L'}{\partial \dot{\theta}_2}\right) = \ddot{\theta}_1\ \left(\frac{2}{3} + \cos(\theta_2)\right) + \ddot{\theta}_2\ 2\left(\frac{1}{3}\right) - \sin(\theta_2)\ \dot{\theta}_1\dot{\theta}_2$$

When we plug all this into Lagranges equation $\frac{d}{dt}\left(\frac{\partial L}{\partial \dot{\theta}_i}\right) - \frac{\partial L}{\partial \theta_i} = T_i$ we get:

$$\ddot{\theta}_1\ 2\left(\frac{5}{3} + \cos(\theta_2)\right) + \ddot{\theta}_2\ \left(\frac{2}{3} + \cos(\theta_2)\right) = \frac{T_1}{\frac{1}{2}ml^2} + \sin(\theta_2)\ \dot{\theta}_2(2\dot{\theta}_1+\dot{\theta}_2)$$

$$\ddot{\theta}_1\ \left(\frac{2}{3} + \cos(\theta_2)\right) + \ddot{\theta}_2\ 2\left(\frac{1}{3}\right) = \frac{T_2}{\frac{1}{2}ml^2} - \sin(\theta_2)\ \dot{\theta}_1^2$$

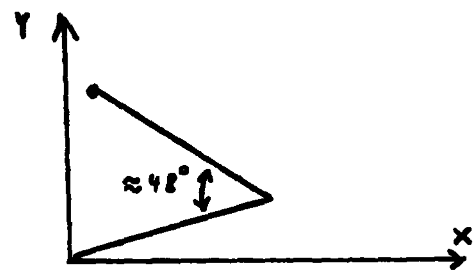
And if you think that was painful, try it the other way! So finally we have a set of equations that allow us to calculate joint-torques given desired joint-accelerations. Notice that we need to know the arm-state, θ_1, θ_2, $\dot{\theta}_1$ and $\dot{\theta}_2$ in order to do this. In part this is because of the appearance of velocity-product terms, representing centrifugal forces and the like, and in part it is because the coefficients of the accelerations vary with the arm-configuration. It is useful to separate out these latter terms which constitute the sensitivity matrix.

$$\begin{vmatrix} 2\left(\frac{5}{3} + \cos(\theta_2)\right) & \left(\frac{2}{3} + \cos(\theta_2)\right) \\ \left(\frac{2}{3} + \cos(\theta_2)\right) & 2\left(\frac{1}{3}\right) \end{vmatrix}$$

If we ignore the velocity-product terms, this matrix tells us the sensitivity of the angular accelerations with respect to the applied torques. It can be shown that the terms in this matrix will depend only on the generalized coordinates (and not the velocities .), that the matrix must be symmetrical and that the diagonal terms must be positive.

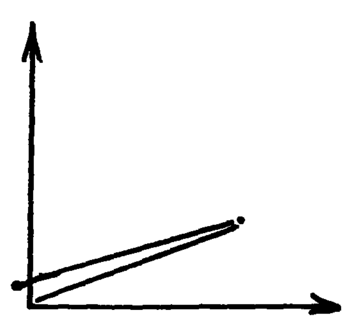
THE SENSITIVITY MATRIX

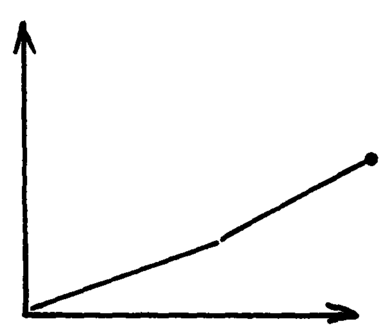
This, by the way, implies that if one makes the torques large enough to over-come the velocity-product terms, the links will move in the expected direction. The analog, positional approach to arm-control depends critically on this property. Notice the couplings between links - that is torque applied to one joint will cause angular accelerations of both links in general. These off-diagonal terms may in fact become negative. In our case, the two joints are decoupled for $\cos(\theta_2) = -2/3$. That is - at least for a moment - torques applied to one joint will only cause angular acceleration of that joint.



In the above diagram, if a torque is applied to joint 1, the push exerted on the end of link 2 is just enough to cause it to have an angular acceleration equal to that of link 1 and so $\ddot{\theta}_2 = 0$. Similarly if a torque is applied to joint 2 it will only cause angular accelerations in θ_2.

Next notice that the diagonal terms vary in size with the arm configuration. It is not surprising that link 1 is most sensitive to torque 1 when $\theta_2 \approx \pi$, and least sensitive for $\theta_2 \approx 0$.





THE INVERSE MATRIX

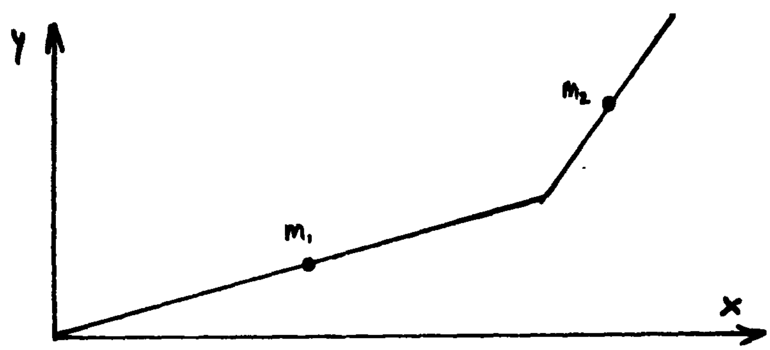
If we wish to know exactly what accelerations will be produced by given torques we have to solve for $\ddot{\theta}_1$ and $\ddot{\theta}_2$ in the above equations.

$$\ddot{\theta}_1 = \left[2(\tfrac{1}{3})\, T_1' \quad - \quad (\tfrac{2}{3} + \cos(\theta_2))\, T_2' \right] / \; (16/9 - \cos^2(\theta_2))$$

$$\ddot{\theta}_2 = \left[-(\tfrac{2}{3} + \cos(\theta_2))\, T_1' \; + 2(\tfrac{5}{3} + \cos(\theta_2))\, T_2' \right] / \; (16/9 - \cos^2(\theta_2))$$

Where $T_1' = \dfrac{T_1}{\frac{1}{2}ml^2} + \sin(\theta_2)\, \dot{\theta}_2(2\dot{\theta}_1+\dot{\theta}_2)$ & $T_2' = \dfrac{T_2}{\frac{1}{2}ml^2} - \sin(\theta_2)\, \dot{\theta}_1^2$

GRAVITY



We can define the potential energy P as the sum of the products of the link masses and the elevation of their center of mass relative to some arbitrary plane.

$$P = g m_1\, \tfrac{1}{2} l_1\, \sin(\theta_1) + g m_2 \left[l_1\, \sin(\theta_1) + \tfrac{1}{2} l_2\, \sin(\theta_1+\theta_2) \right]$$

We could now repeat the above calculation, subtracting this term from the kinetic energy. Because of the linearity of the equations, we can again make use of superposition and calculate the torques required to balance gravity separately. Now the partial derivatives of P with respect to the angular velocities are 0 so we only need the following:

$$T_{1g} = \frac{\partial P}{\partial \theta_1} = g \left[(\tfrac{1}{2}m_1+m_2)\, l_1\, \cos(\theta_1) + \tfrac{1}{2}m_2 l_2\, \cos(\theta_1+\theta_2) \right]$$

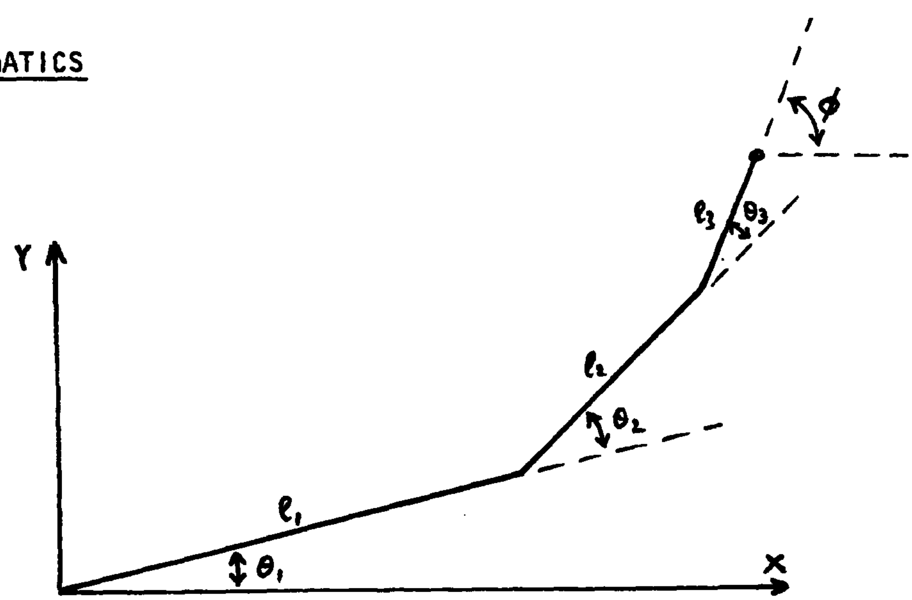
$$T_{2g} = \frac{\partial P}{\partial \theta_2} = g \left[\tfrac{1}{2}m_2 l_2\, \cos(\theta_1+\theta_2) \right]$$

as before

THREE-LINK MANIPULATOR:

A manipulator not only has to be able to reach points within a given work-space it also has to be able to approach the object to be manipulated with various orientations of the terminal device. That is,we need a position and orientation generator. Similarly it can be argued that it should not only be able to apply forces to the object, but torques as well. Additional degrees of freedom are required to accomplish this. If we are confined to operation in a two-dimensional space only one extra degree of freedom will be needed; since rotation can take place only about one axis, the axis normal to the plane of operation. It turns out that the same can be said about torque, since applying a torque can be thought of as an attempt to cause a rotation. So in two dimensions, a three link manipulator is sufficient for our purposes. We will now repeat our analysis of kinematics, statics and dynamics for this device - with fewer details than before.

KINEMATICS



$$x = l_1 \cos(\theta_1) + l_2 \cos(\theta_1+\theta_2) + l_3 \cos(\theta_1+\theta_2+\theta_3)$$

$$y = l_1 \sin(\theta_1) + l_2 \sin(\theta_1+\theta_2) + l_3 \sin(\theta_1+\theta_2+\theta_3)$$

$$\phi = \theta_1+\theta_2+\theta_3$$

As before we could now proceed to solve the inverse problem of finding joint-angles from tip-position and orientation by geometric, trigonometric or algebraic methods. This is not much harder than it was for two links since one can use the equation for ϕ to eliminate one of the three joint-angles from the other two equations and so has, again, only two trigonometric polynomials to solve. This is left as an exercise!

It is simpler to make use of the results for the two-link manipulator, since one can easily calculate the position of joint 2, knowing ϕ:

$$x_2 = x - l_3 \cos(\phi) \qquad \text{and} \qquad y_2 = y - l_3 \sin(\phi)$$

Now one can simply solve the remaining two-link device precisely as before:

$$\cos(\theta_2) = \frac{(x_2^2+y_2^2)-(l_1^2+l_2^2)}{2\; l_1\; l_2}$$

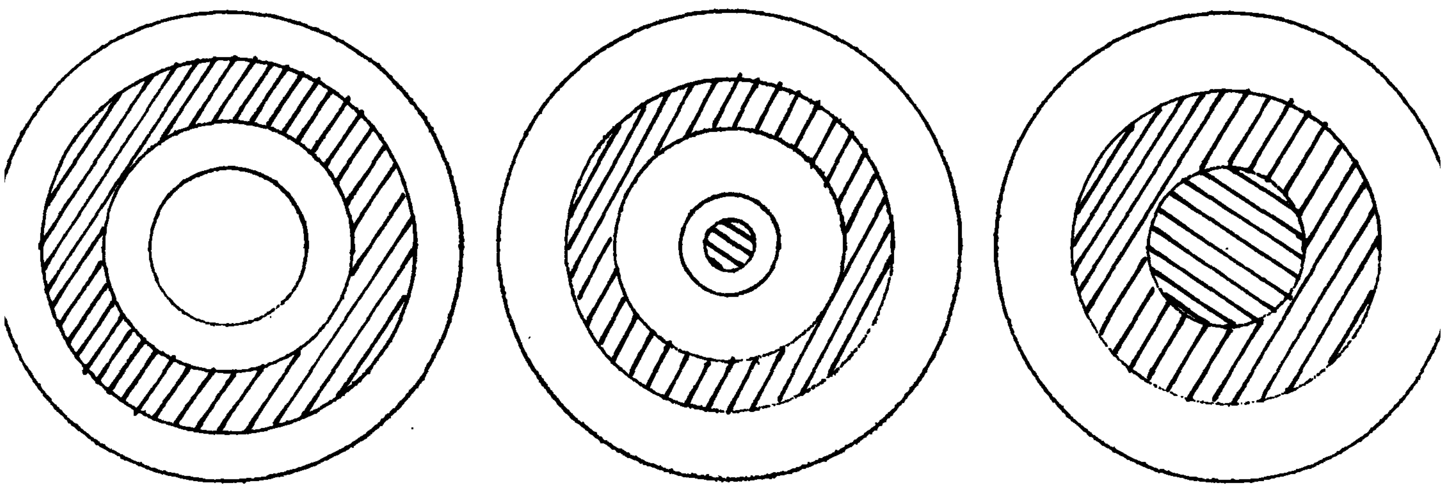
$$\tan(\theta) = y_2/x_2 \qquad \tan(\alpha) = \frac{l_2 \sin(\theta_2)}{l_1 + l_2 \cos(\theta_2)} = \frac{2\; l_1\; l_2 \sin(\theta_2)}{(x_2^2+y_2^2) + (l_1^2-l_2^2)}$$

$$\theta_1 = \theta - \alpha \qquad \text{and finally} \quad \theta_3 = \phi - \theta_2 - \theta_1$$

To determine how much of the work-space that can be reached by the manipulator is usable with arbitrary orientation of the last link, we could, as before, proceed with an algebraic approach. For example we might start from $|\cos(\theta_2)| \leq 1$ and the realization that the worst case situations occur when the last link is parallel to the direction from the origin to the tip (that is $\cos(\phi) = \pm x/\sqrt{x^2+y^2}$ and $\sin(\phi) = \pm y/\sqrt{x^2+y^2}$). The situation is easy enough to visualize, so we will use geometric reasoning instead.

USABLE WORK-SPACE

Not all points in the annular work-space previously determined can be reached with arbitrary orientation of the last link, A method for constructing the usable work-space is simply to construct a circle of radius l_3 about each point. A point is in the usable work-space if the circle so constructed lies inside the annulus previously determined.

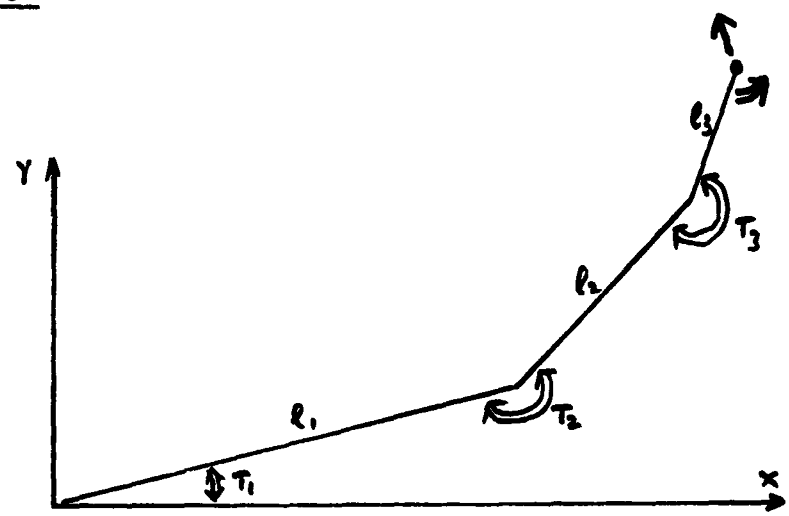


$l_3 < |l_1 - l_2|$ $\quad$ $l_3 > |l_1 - l_2|$ $\quad$ $l_1 = l_2$

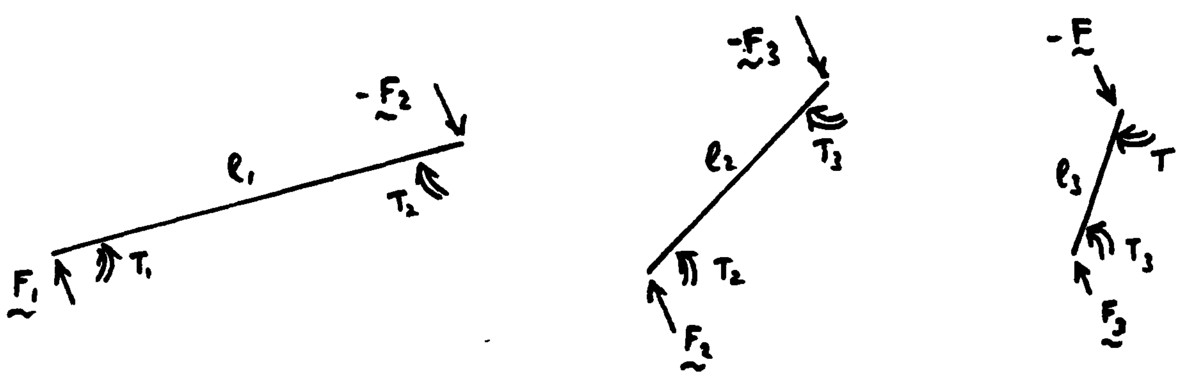
If l_3 is less than $|l_1 - l_2|$, this new region is again an annulus, with inner radius $l_3 + |l_1 - l_2|$ and outer radius $l_1 + l_2 - l_3$. The width of the annulus is twice the length of the shorter link minus l_3, its average radius is still equal to the length of the longer of link 1 and link 2. It is obviously a good idea to keep the third link short in order to achieve a reasonably large usable work-space.

If l_3 is greater than $|l_1 - l_2|$, there is an additional circular region centered on the origin of radius $l_3 - |l_1 - l_2|$. The circular and annular regions merge when $l_1 = l_2 = l$ say, and form one circular region of radius $2\,l - l_3$. The advantages of having the first two links of equal length again become apparent.

STATICS:



We have control over the three torques T_1, T_2 and T_3 and would like to use these to apply force $\underline{F}=(u,v)$ and torque T to the object held by the tip of the device. We do not want to consider motion of the manipulator now, so imagine its tip solidly fixed in place. We proceed by writing down one equation for force balance for each link and one equation for torque balance for each link.



$\underline{F}_1 = \underline{F}_2$, $\underline{F}_2 = \underline{F}_3$ and $\underline{F}_3 = \underline{F}$ so $\underline{F} = \underline{F}_1 = \underline{F}_2 = \underline{F}_3$

$T_3 = T + \underline{r}_3 \times \underline{F}$, $T_2 = T_3 + \underline{r}_2 \times \underline{F}_3$ and $T_1 = T_2 + \underline{r}_1 \times \underline{F}_2$

$$T_1 = T + (\underline{r}_1+\underline{r}_2+\underline{r}_3) \times \underline{F}$$
$$T_2 = T + (\underline{r}_2+\underline{r}_3) \times \underline{F}$$
$$T_3 = T + (\underline{r}_3) \times \underline{F}$$

Let's abbreviate the trigonometric terms, for example $s_{23} = \sin(\theta_2+\theta_3)$, then

$$\underline{r}_1 = l_1(c_1,s_1)\ ,\quad \underline{r}_2 = l_2(c_{12},s_{12}) \quad \text{and } \underline{r}_3 = l_3(c_{123},s_{123})$$

and so:
$$(\underline{r}_3) \times \underline{F} = (l_3c_{123})\ v - (l_3s_{123})\ u$$
$$(\underline{r}_2+\underline{r}_3) \times \underline{F} = (l_2c_{12}+l_3c_{123})\ v - (l_2s_{12}+l_3s_{123})\ u$$
$$(\underline{r}_1+\underline{r}_2+\underline{r}_3) \times \underline{F} = (l_1c_1+l_2c_{12}+l_3c_{123})\ v - (l_1s_1+l_2s_{12}+l_3s_{123})\ u$$

$$\begin{vmatrix} T_1 \\ T_2 \\ T_3 \end{vmatrix} = \begin{vmatrix} -(l_1s_1+l_2s_{12}+l_3s_{123}) & (l_1c_1+l_2c_{12}+l_3c_{123}) & 1 \\ -(l_2s_{12}+l_3s_{123}) & (l_2c_{12}+l_3c_{123}) & 1 \\ -(l_3s_{123}) & (l_3c_{123}) & 1 \end{vmatrix} \begin{vmatrix} u \\ v \\ T \end{vmatrix}$$

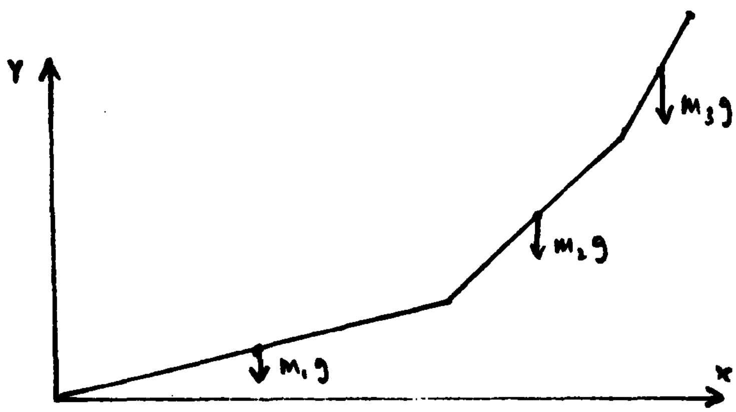
So we can easily calculate what motor-torques are needed to apply a given force and torque to the object. Notice that we could have arrived at this result by first considering the tip pinned in place only, that is T = 0, and then separately reason out that to apply torque T, each joint-torque would have to be increased by T.

The determinant of the above matrix is $l_1\ l_2\ \sin(\theta_2)$. So if $\theta_2 \neq 0$ or π, we can invert the matrix and solve for u, v and T given the three joint-torques.

$$\begin{vmatrix} u \\ v \\ T \end{vmatrix} = \begin{vmatrix} l_2c_{12} & -(l_1c_1+l_2c_{12}) & l_1c_1 \\ l_2s_{12} & -(l_1s_1+l_2s_{12}) & l_1s_1 \\ l_2l_3s_3 & -(l_1l_3s_{23}+l_2l_3s_3) & l_1l_2s_2+l_1l_3s_{23} \end{vmatrix} \begin{vmatrix} T_1 \\ T_2 \\ T_3 \end{vmatrix} \Big/ l_1l_2s_2$$

GRAVITY:

Gravity is again very simple to take into account. If we assume that the center of mass of each link is in its geometric center we find that:



$\underline{F}_3 = m_3\underline{g}$, $\underline{F}_2 = (m_2+m_3)\underline{g}$ and $\underline{F}_1 = (m_1+m_2+m_3)\underline{g}$ and so:

$$T_{3g} = - m_3 \tfrac{1}{2}\underline{r}_3 \times \underline{g} = g\left[\tfrac{1}{2}m_3 l_3 c_{123}\right]$$

$$T_{2g} = T_{3g} - m_2 \tfrac{1}{2}\underline{r}_2 \times \underline{g} - m_3 \underline{r}_2 \times \underline{g}$$

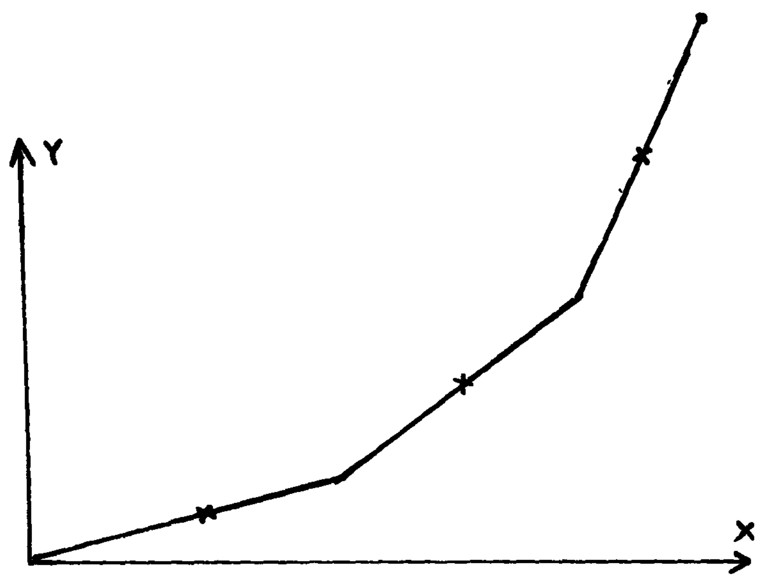
$$= g\left[(\tfrac{1}{2}m_2+m_3) l_2 c_{12} + \tfrac{1}{2}m_3 l_3 c_{123}\right]$$

$$T_{1g} = T_{2g} - m_1 \tfrac{1}{2}\underline{r}_1 \times \underline{g} - (m_2+m_3)\, \underline{r}_1 \times \underline{g}$$

$$= g\left[(\tfrac{1}{2}m_1+m_2+m_3) l_1 c_1 + (\tfrac{1}{2}m_2+m_3) l_2 c_{12} + \tfrac{1}{2}m_3 l_3 c_{123}\right]$$

DYNAMICS:

For definiteness we will again consider a simple case where $l_1 = l_2 = l_3 = l$ say. The more general case involves a lot more arithmetic and the form of the final result is the same, only numerical constant will be changed. Further, we will ignore gravity for now, and assume the links to be uniform sticks of mass m and inertia $(1/12)ml^2$ about their center of mass.



Once again we start by finding the rotational and translational velocities of each of the links. Evidently the angular velocities of the three links are $\dot\theta_1$, $(\dot\theta_1+\dot\theta_2)$ and $(\dot\theta_1+\dot\theta_2+\dot\theta_3)$.

The square of the magnitude of the instantaneous linear velocity of the center of mass of link 1 is simply:

$$|\tfrac{1}{2}\underline{r}_1\dot\theta_1|^2 = l^2(\tfrac{1}{4}\dot\theta_1^2)$$

For the square of the magnitude of the velocity of the center of link 2:

$$|\underline{r}_1\dot\theta_1 + \tfrac{1}{2}\underline{r}_2(\dot\theta_1+\dot\theta_2)|^2 = l^2[\dot\theta_1^2 + \cos(\theta_2)\dot\theta_1(\dot\theta_1+\dot\theta_2) + \tfrac{1}{4}(\dot\theta_1+\dot\theta_2)^2]$$

$$= l^2[\dot\theta_1^2(\tfrac{5}{4} + \cos(\theta_2)) + \dot\theta_1\dot\theta_2(\tfrac{1}{2} + \cos(\theta_2)) + \dot\theta_2^2\ (\tfrac{1}{4})]$$

For the square of the magnitude of the velocity of the center of link 3:

$$|\underline{r}_1\dot\theta_1 + \underline{r}_2(\dot\theta_1+\dot\theta_2) + \tfrac{1}{2}\underline{r}_3(\dot\theta_1+\dot\theta_2+\dot\theta_3)|^2 = 1^2\Big[\dot\theta_1^2+2\cos(\theta_2)\dot\theta_1(\dot\theta_1+\dot\theta_2)$$
$$+(\dot\theta_1+\dot\theta_2)^2 + \cos(\theta_3)(\dot\theta_1+\dot\theta_2)(\dot\theta_1+\dot\theta_2+\dot\theta_3) + \tfrac{1}{4}(\dot\theta_1+\dot\theta_2+\dot\theta_3)^2$$
$$+ \cos(\theta_2+\theta_3)\dot\theta_1(\dot\theta_1+\dot\theta_2+\dot\theta_3)\Big]$$
$$= 1^2\Big[\dot\theta_1^2\left(\tfrac{9}{4} + 2\cos(\theta_2) + \cos(\theta_3) + \cos(\theta_2+\theta_3)\right)$$
$$+\dot\theta_1\dot\theta_2\left(\tfrac{11}{2} + 2\cos(\theta_2) + 2\cos(\theta_3) + \cos(\theta_2+\theta_3)\right)$$
$$+ \dot\theta_2^2\left(\tfrac{5}{4} + \cos(\theta_3)\right)$$
$$+\dot\theta_2\dot\theta_3\left(\tfrac{1}{2} + \cos(\theta_3)\right)$$
$$+ \dot\theta_3^2\left(\tfrac{1}{4}\right)$$
$$+ \dot\theta_3\dot\theta_1\left(\tfrac{1}{2} + \cos(\theta_3) + \cos(\theta_2+\theta_3)\right)\Big] \qquad \text{(ugh!)}$$

We are now ready to add up the kinetic energy due to rotation and that due to linear translation of the center of mass for all three links.

$$L = \tfrac{1}{2}ml^2\Big[\dot\theta_1^2\left(4 + 3\cos(\theta_2) + \cos(\theta_2+\theta_3) + \cos(\theta_3)\right)$$
$$+\dot\theta_1\dot\theta_2\left(\tfrac{19}{3} + 3\cos(\theta_2) + \cos(\theta_2+\theta_3) + 2\cos(\theta_3)\right)$$
$$+ \dot\theta_2^2\left(\tfrac{5}{3} + \cos(\theta_3)\right)$$
$$+\dot\theta_2\dot\theta_3\left(\tfrac{2}{3} + \cos(\theta_3)\right)$$
$$+ \dot\theta_3^2\left(\tfrac{1}{3}\right)$$
$$+\dot\theta_3\dot\theta_1\left(\tfrac{2}{3} + \cos(\theta_2+\theta_3) + \cos(\theta_3)\right)\Big]$$

So this is the lagrangian for this system and from it we will be able to calculate the relation between joint-torques and joint-accelerations. Let us use the short-hand notation for trigonometric terms introduced in the discussion of the statics of this device.

We will next derive the partial derivates of the Lagrangian with respect to θ_1, θ_2, θ_3, $\dot{\theta}_1$, $\dot{\theta}_2$ and $\dot{\theta}_3$. Let $L = \frac{1}{2}ml^2 L'$ as before.

$$-\frac{\partial L'}{\partial \theta_1} = 0$$

$$-\frac{\partial L'}{\partial \theta_2} = \dot{\theta}_1^2(3\, s_2 + s_{23}) + \dot{\theta}_1\dot{\theta}_2(3\, s_2 + s_{23}) + \dot{\theta}_3\dot{\theta}_1(s_{23})$$

$$-\frac{\partial L'}{\partial \theta_3} = \dot{\theta}_1^2(s_{23}+s_3) + \dot{\theta}_1\dot{\theta}_2(s_{23}+2\, s_3) + \dot{\theta}_2^2(s_3) + \dot{\theta}_2\dot{\theta}_3(s_3) + \dot{\theta}_3\dot{\theta}_1(s_{23}+s_3)$$

$$\frac{\partial L'}{\partial \dot{\theta}_1} = 2\dot{\theta}_1(4+3c_2+c_{23}+c_3) + \dot{\theta}_2(\tfrac{19}{3}+3c_2+c_{23}+2c_3) + \dot{\theta}_3(\tfrac{2}{3}+c_{23}+c_3)$$

$$\frac{\partial L'}{\partial \dot{\theta}_2} = \dot{\theta}_1(\tfrac{19}{3}+3c_2+c_{23}+2c_3) + 2\dot{\theta}_2(\tfrac{5}{3}+c_3) + \dot{\theta}_3(\tfrac{2}{3}+c_3)$$

$$\frac{\partial L'}{\partial \dot{\theta}_3} = \dot{\theta}_1(\tfrac{2}{3}+c_{23}+c_3) + \dot{\theta}_2(\tfrac{2}{3}+c_3) + 2\dot{\theta}_3(\tfrac{1}{3})$$

Next we will need the time rate -of-change of the last three quantities above:

$$\frac{d}{dt}\left(\frac{\partial L'}{\partial \dot{\theta}_1}\right) = 2\ddot{\theta}_1(4+3c_2+c_{23}+c_3) + \ddot{\theta}_2(\tfrac{19}{3}+3c_2+c_{23}+2c_3) + \ddot{\theta}_3(\tfrac{2}{3}+c_{23}+c_3)$$
$$- 2\dot{\theta}_1(3s_2\dot{\theta}_2-s_{23}(\dot{\theta}_2+\dot{\theta}_3)-s_3\dot{\theta}_3) - \dot{\theta}_2(3s_2\dot{\theta}_2+s_{23}(\dot{\theta}_2+\dot{\theta}_3)-2s_3\dot{\theta}_3)$$
$$- \dot{\theta}_3(s_{23}(\dot{\theta}_2+\dot{\theta}_3)-s_3\dot{\theta}_3)$$

$$\frac{d}{dt}\left(\frac{\partial L'}{\partial \dot{\theta}_2}\right) = \ddot{\theta}_1(\tfrac{19}{3}+3c_2+c_{23}+2c_3) + 2\ddot{\theta}_2(\tfrac{5}{3}+c_3) + \ddot{\theta}_3(\tfrac{2}{3}+c_3)$$
$$- \dot{\theta}_1(3s_2\dot{\theta}_2-s_{23}(\dot{\theta}_2+\dot{\theta}_3)-s_3\dot{\theta}_3) - 2\dot{\theta}_2(s_3\dot{\theta}_3) - \dot{\theta}_3(s_3\dot{\theta}_3)$$

$$\frac{d}{dt}\left(\frac{\partial L'}{\partial \dot{\theta}_3}\right) = \ddot{\theta}_1(\tfrac{2}{3}+c_{23}+c_3) + \ddot{\theta}_2(\tfrac{2}{3}+c_3) + 2\ddot{\theta}_3(\tfrac{1}{3})$$
$$- \dot{\theta}_1(s_{23}(\dot{\theta}_2+\dot{\theta}_3)-s_3\dot{\theta}_3) - \dot{\theta}_2(s_3\dot{\theta}_3)$$

Finally, inserting these terms into Lagranges equation:

$$T_1' = \frac{d}{dt}\left(\frac{\partial L'}{\partial \dot{\theta}_1}\right) - \frac{\partial L'}{\partial \theta_1} = 2\ddot{\theta}_1(4+3c_2+c_{23}+c_3) + \ddot{\theta}_2\left(\frac{19}{3} + 3c_2+c_{23}+2c_3\right) + \ddot{\theta}_3\left(\frac{2}{3}+c_{23}+c_3\right)$$

$$- \dot{\theta}_1\dot{\theta}_2(6s_2+2s_{23}) - \dot{\theta}_2^2(3s_2+s_{23}) - \dot{\theta}_2\dot{\theta}_3(2s_{23}) - \dot{\theta}_3^2(s_{23}+s_3) - \dot{\theta}_3\dot{\theta}_1(2s_{23}+s_3)$$

$$T_2' = \frac{d}{dt}\left(\frac{\partial L'}{\partial \dot{\theta}_2}\right) - \frac{\partial L'}{\partial \theta_2} = \ddot{\theta}_1\left(\frac{19}{3}+3c_2+c_{23}+2c_3\right) + 2\ddot{\theta}_2\left(\frac{5}{3}+c_3\right) + \ddot{\theta}_3\left(\frac{2}{3}+c_3\right)$$

$$+ \dot{\theta}_1^2(3s_2+s_{23}) - \dot{\theta}_2\dot{\theta}_3(2s_3) - \dot{\theta}_3^2(s_3) - \dot{\theta}_3\dot{\theta}_1(2s_3)$$

$$T_3' = \frac{d}{dt}\left(\frac{\partial L'}{\partial \dot{\theta}_3}\right) - \frac{\partial L'}{\partial \theta_3} = \ddot{\theta}_1\left(\frac{2}{3}+c_{23}+c_3\right) + \ddot{\theta}_2\left(\frac{2}{3}+c_3\right) + \ddot{\theta}_3\left(\frac{2}{3}\right)$$

$$+ \dot{\theta}_1^2(s_{23}+s_3) + \dot{\theta}_1\dot{\theta}_2(2s_3) + \dot{\theta}_2^2(s_3)$$

$$\begin{vmatrix} T_1 \\ T_2 \\ T_3 \end{vmatrix} = \tfrac{1}{2}ml^2 \begin{vmatrix} 2(4+3c_2+c_{23}+c_3) & \left(\frac{19}{3}+3c_2+c_{23}+2c_3\right) & \left(\frac{2}{3}+c_{23}+c_3\right) \\ \left(\frac{19}{3}+3c_2+c_{23}+2c_3\right) & 2\left(\frac{5}{3}+c_3\right) & \left(\frac{2}{3}+c_3\right) \\ \left(\frac{2}{3}+c_{23}+c_3\right) & \left(\frac{2}{3}+c_3\right) & 2\left(\frac{1}{3}\right) \end{vmatrix} \begin{vmatrix} \ddot{\theta}_1 \\ \ddot{\theta}_2 \\ \ddot{\theta}_3 \end{vmatrix} + \begin{vmatrix} T_{1v} \\ T_{2v} \\ T_{3v} \end{vmatrix}$$

Where T_{1v}, T_{2v} and T_{3v} are velocity product terms, $\frac{1}{2}ml^2$ times the second lines in each of the expansions above for T_1', T_2' and T_3'.

Notice once again the symmetry of the sensitivity matrix and the fact that its diagonal elements are always positive. Also remember that the terms in this matrix can depend only on the joint-angles, all velocity-product terms being segregated out on the right.

Clearly then, given the arm-state (θ_1, θ_2, θ_3, $\dot{\theta}_1$, $\dot{\theta}_2$ and $\dot{\theta}_3$), we can calculate what torques need to be applied to each of the joints in order to achieve a given angular acceleration for each of the joints.

EXTENSIONS TO THREE DIMENSIONS:

Once the basic principles are understood, we can proceed to introduce the extensions necessary to deal with manipulators in three dimensions. There is little difficulty as regards position and force since in an n-dimensional space these quantities can be conveniently represented by n-dimensional vectors. A general position or force generator will need n degrees of freedom. Unfortunately we are not so lucky with orientation and torque. These can not be usefully thought of as vectors. For example, in three dimensions we know that rotations don't commute, while vector addition does. It is a misleading coincidence that it takes three variables to specify a rotation in three dimensions.

ROTATION:

It takes $n(n-1)/2 = \binom{n}{2}$ variables to specify a rotation in n dimensional space. Why? A general rotation can be made up of components each of which carries one axis part way towards a second axis. There are n axes and so $n(n-1)/2$ distinct pairs of axes and therefore that number of "elementary" rotations. It is not correct to think of rotations "about an axis"; in our two dimensional example such rotations would carry one out of the plane of the paper, and in four dimensions, not all possible rotations would be generated by considering only combinations of the four rotations about the coordinate axes.

Another way of approaching this problem is to look at matrices that represent coordinate transformations that correspond to rotations. Such matrices are ortho-normal and of size n x n. How many of the n^2 entries can be freely chosen? The condition of normality generates n constraints, and the condition of orthogonality another $n(n-1)/2$. So we have $n^2 - n - n(n-1)/2 = n(n-1)/2$ degrees of freedom left.

To specify position and orientation or force and torque in n dimensions requires $n(n-1)/2 + n = n(n+1)/2$ variables. A general-purpose n dimensional manipulator needs to have $n(n+1)/2$ degrees of freedom. For $n = 3$, this is 6.

The coincidence that it takes 3 variables to specify a rotation in three dimensions allows some simplifications - a torque for example can be calculated by taking cross-products. In higher dimensions, one needs to look at exterior tensor products. A useful way of specifying rotations in three dimensions is by means of Euler angles - roll, pitch and yaw for example. It is straightforward to convert between this representation and the ortho-normal matrix notation.

KINEMATICS:

It is no longer sufficient to represent each link as a vector, since the joints at its two ends may have axes that are not parallel. The way to deal with this problem is simply to erect a coordinate system fixed to each link. Corresponding to each joint then there will be a coordinate transformation from one system to the next. This transformation can be represented by a 3 x 3 rotation matrix plus a 3 x 1 offset vector. It is convenient to combine these into one 4 x 4 transformation matrix that has (0 0 0 1) as its last row. This allows one easily to invert the transformation, so as to allow convertion of coordinates in the other direction as well.

The entries in this matrix will be trigonometric polynomials in the joint-angles. In order to determine the relation between links separated by more than one joint, one can simply multiply the transformation matrices corresponding to the intervening joints. Doing this for the complete manipulator, one obtains a single matrix that allows one to relate coordinates relative to the tip or terminal device to coordinates relative to the base of the device. In fact the 3 x 3 rotation submatrix gives us the rotation of the last link relative to the base and hence its orientation, while the offset 3 x 1 submatrix is the position of the tip of the last link with respect to the base.

Given the joint-variables, it is then a relatively straight-forward matter to arrive at the position and orientation of the terminal device or tip. These values are of course unique for a particular set of joint-variables.

THE INVERSE PROBLEM:

Unfortunately the inversion is much harder. One way to approach this problem would be to consider the 3 x 3 rotation sub-matrix made up entirely of polynomials in sines and cosines of joint-angles and the 3 x 1 offset sub-matrix which contains link-length as well and try to solve for the sines and cosines of the six joint-angles. There are twelve equations in twelve unknowns, so we expect there to be a finite number of solutions. When solving polynomial equations by eliminating variables the degree of the resulting polynomials grows as the product of the polynomials combined. We could easily end up with one polynomial in one unknown with a degree of several thousand. So in general this problem is intractable.

There are a number of conditions on the link geometry that make this problem solvable by non-iterative techniques. Several such configurations are known, but one of the easiest to explain involves decoupling the orientation from the position. One then has to solve two problems that are much smaller, each having only three degrees of freedom. Suppose for example that the last three rotational joints intersect in one point, call it the wrist. Then these last three can take care of the orientation, while the remaining three position the wrist. Given the orientation of the last link it is easy to calculate where the wrist should be relative to the tip position. Given the position of the wrist one can solve the inversion problem for the first three links.This can usually be done by careful inspection rather than blind solution of trigonometric polynomials. Often also the first three links are simply a combination of the two-link geometry we have already solved and an offset polar-coordinate problem.

Now that we know the first three joint-angles we can calculate the orientation of the third to which the wrist attaches. Comparing this with the desired orientation of the last link, it is simple to calculate the three wrist-angles by matrix multiplication and solving for the Euler angles appropriate to the design of the wrist.

STATICS:

By controlling the six joint-torques we can produce a given force and torque at the terminal device. The same coordinate transformation matrices used for solving the kinematics prove useful here. Cross-products give us the required torques, with joint-motors supporting the components around the joint-axes, while the pin-joints transmit the other components. The calculations are straight-forward.

Gravity compensation calculations also follow the familiar pattern. In many cases manipulators intended for positional control have been used to generate forces and torques in a different manner. The idea is to use the inherent compliance of the device as a kind of spring and to drive the joints to angles slightly away from the equilibrium position. Since the stress-strain matrix of such a device is very complex and it has different springconstants in different directions, as well as coupling between forces and torques, this technique on its own is not very useful. One solution relies on a force and torque sensor in the wrist. From the output of such a device one can calculate the forces and torques at the tip and servo the joint-angles accordingly. The advantage of this technique is that friction in the first three joints does not corrupt the result and that the measurement is made beyond the point where the heaviest and stickiest components of the manipulator are.

DYNAMICS:

The main additional difficulty of manipulators in higher dimensions is that inertia too now has several components instead of just one. The dynamic behaviour of a rigid body as regards rotation can be conveniently expressed as a symmetrical,square inertia matrix. This relates the applied torque components to the resulting angular accelerations. The same general idea carries through, with the distinction that the calculations get very messy and have to be approached in a systematic fashion. A practical difficulty is the measurement of the components of the inertia matrices for each of the links of the manipulator.

FURTHER READING:

For a detailed analysis of the kinematics of a real three dimensional manipulator:

"KINEMATICS OF THE MIT-AI-VICARM MANIPULATOR", B.K.P.Horn and H.Inoue, MIT-AI-WP-69, May '74

This paper also discusses such things as Euler angles in more detail, gives the transformation matrices from link to link, and has lots of useful references. For a concise account of some of the best work with trajectory control of manipulators see:

"TRAJECTORY CONTROL OF A COMPUTER ARM", R. Paul,3IJCAI, pp385-390

More details are available in:

"MODELLING, TRAJECTORY CALCULATION AND SERVOING OF A COMPUTER CONTROLLED ARM", R. Paul, Stanford-AIM-177, '72

The following form a sequence that lead up to proper understanding:

"ON THE DYNAMIC ANALYSIS OF SPATIAL LINKAGES USING 4 x 4 MATRICES" J.J. Uicker, Ph.D. Dissertation, Northwestern University, Evanstan, Illinois, Aug '65

"DYNAMIC FORCE ANALYSIS OF SPATIAL LINKAGES", J.J. Uicker, Transactions ASME, '67

"THE KINEMATICS OF MANIPULATORS UNDER COMPUTER CONTROL" D.L. Pieper, Stanford-AIM-72 '68

"THE NEAR-MINIMUM-TIME CONTROL OF OPEN-LOOP ARTICULATED KINEMATIC CHAINS", M.E. Kahn, Stanford-AIM-106, '69